

## 1. Product Introduction

We offer the Bancord round belt (joint type) of a long cord type as a round belt. However, for recent office equipment and optical machinery, the needs for round belts that do not require joining and have excellent low-temperature characteristics have been increasing. The Bancollan round belt is a high-performance round belt that has undergone our original quality improvements and meets those needs.

### Features

#### Easy belt installation

The belt can be easily installed by stretching it even when the center distance is fixed. Unlike a belt containing tension members, there is no need to slide pulleys or take time for tension adjustment; hence, it reduces the installation man-hour.

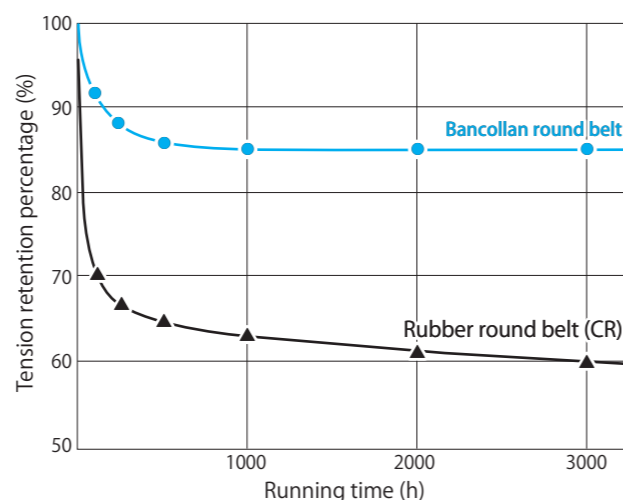
#### Stable tension

For belts without tension members, tension stability is especially important for belt performance. The Bancollan round belt has less changes in tension due to bending or permanent elongation than general rubber round belts or belts containing tension members, and can be used with almost no maintenance.

#### Smooth start even at low temperature

The specially compounded polyurethane rubber mostly prevents the belt from hardening or becoming set even at -20 °C. Therefore, it starts smoothly with no trouble due to the starting torque.

Fig. 1 Tension changes of the Bancollan round belt (Initial stretch rate 6%)



### Example of use

#### Office equipment

- Copiers
- Fax machines
- Electric typewriters
- Registers
- Ticket vending machines, bill exchange machines
- Automatic ticket gates
- Weighing-pricing machines
- Automatic cash payment machines

#### Optical equipment/music equipment

- Tape decks
- VTR

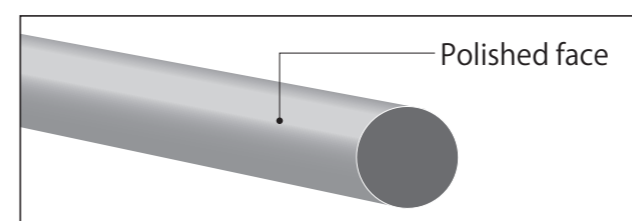
#### Others

- Ultra-compact fans, air pollution measurement machines
- Vacuum cleaners, spectroscopic analysis devices
- Stirring machines, desktop winding machines
- Roller conveyors
- Rotating lights
- Polishing machines

### Structure and Characteristics

For orders for Bancollan round belts, please specify the belt sizes shown in Table 1 if possible. However, as this belt is made to order, please contact us about delivery period, lots, and prices.

#### Belt appearance



#### Belt indication example

4 × 305

Effective length (mm)

Cross-sectional diameter (mm)

Sizes are indicated in units of individual package.

#### Basic physical properties of the belt

Item	Material	#267WLS
Hardness		72° (JIS)
Specific gravity		1.26
Tensile stress	When stretched by 4%	$35 \times 10^4$ (Pa)
	When stretched by 6%	$50 \times 10^4$ (Pa)
	When stretched by 8%	$64 \times 10^4$ (Pa)
Tensile strength		3000 or more (N/cm <sup>2</sup> )
Elongation at the time of break		600(%)

### Dimensional Tolerance

#### Belt cross-sectional tolerance

(Unit: mm)

Cross-sectional diameter	2	3	4	5
Tolerance	±0.10	±0.10	±0.15	±0.20

#### Effective length

(Unit: mm)

Effective length	Tolerance
100~200	±2.0
201~400	±3.0
401~500	±4.0

### Belt Size

Table 1 Table of belt standard sizes

(Unit: mm)

Effective length	φ2		φ3		φ4		φ5	
	Effective center perimeter	Effective length	Effective center perimeter	Effective length	Effective center perimeter	Effective length	Effective center perimeter	
104	110.2	107	113.4	135	143.1	157	166.4	
107	113.4	108	114.5	140	148.4	168	178.1	
100	106.0	113	119.8	142	150.5	182	192.9	
112	118.7	115	121.9	160	169.6	200	212.0	
115	121.9	120	127.2	165	174.9	210	222.6	
120	127.2	128	135.7	170	180.2	220	233.2	
125	132.5	132	139.9	173	183.4	225	238.5	
130	137.8	138	146.3	175	185.5	230	243.8	
134	142.0	140	148.4	184	195.0	247	261.8	
135	143.1	145	153.7	197	208.8	248	262.9	
140	148.4	150	159.0	200	212.0	250	265.0	
145	153.7	153	162.2	206	218.4	275	291.5	
147	155.8	155	164.3	213	225.8	290	307.4	
152	161.1	160	169.6	225	238.5	300	318.0	
158	167.5	165	174.9	230	243.8	305	323.3	
160	169.6	170	180.2	235	249.1	310	328.6	
163	172.8	172	182.3	240	254.4	315	333.9	
167	177.0	175	185.5	250	265.0	320	339.2	
170	180.2	180	190.8	254	269.2	330	349.8	
180	190.8	182	192.9	258	273.5	345	365.7	
183	194.0	190	201.4	264	279.8	348	368.9	
190	201.4	193	204.6	270	286.2	363	384.8	
194	205.6	200	212.0	275	291.5	374	396.4	
196	207.8	201	213.1	284	301.0	375	397.5	
200	212.0	204	216.2	285	302.1	380	402.8	
213	225.8	213	225.8	290	307.4	402	426.1	
227	240.6	223	236.4	300	318.0	416	441.0	
239	253.3	230	243.8	305	323.3	422	447.3	
244	258.6	236	250.2	316	335.0	434	460.0	
250	265.0	240	254.4	323	342.4	440	466.4	
273	289.4	250	265.0	332	351.9	460	487.6	
290	307.4	260	275.6	346	366.8			
330	349.8	270	286.2	361	382.7			
444	470.6	275	291.5	367	389.0			
		282	298.9	374	396.4			
		285	302.1	377	399.6			
		290	307.4	385	408.1			
		294	311.6	390	413.4			
		305	323.3	415	439.9			
		308	326.5	440	466.4			
		314	332.8	450	477.0			
		318	337.1					
		330	349.8					
		347	367.8					
		356	377.4					
		359	380.5					
		363	384.8					
		376	398.6					
		387	410.2					
		390	413.4					
		395	418.7					
		400	424.0					
		410	434.6					
		430	455.8					
		441	467.5					
		450	477.0					
		460	487.6					

- The effective length is the center perimeter with no stretch.  
 - The effective center perimeter represents the center perimeter when the belt is stretched by 6%.

## 2. How to Design

### Step 1. Determining conditions required for the design

- ① Machine type
- ② Transmission power, or rated power of the driving machine
- ③ Degree of load fluctuation
- ④ Daily operating hours
- ⑤ Speed ratio

$$\left( \frac{\text{Pinion revolution}}{\text{Revolution of large pulley}} \right)$$

- ⑥ Temporary center distance
- ⑦ Pulley diameter restriction
- ⑧ Operating environment (high temperature, low temperature, oil, water, dirt, acid, alkali)

### Step 2. Calculating the design power

Correct the driven load and obtain the design power to be used.

$$P_d = P_t \times \left( \frac{K_o}{K_{\theta_1} \times K_t} \right)$$

$P_d$  : Design power (W)  
 $P_t$  : Transmission power (driven load or motor rating) (W)  
 $K_o$  : Load correction factor (Table 2)  
 $K_{\theta_1}$  : Pinion contact angle correction factor (Table 3)  
 $K_t$  : Correction factor by initial stretch rate (Table 4)

When the transmission power was given in torques, convert it into watts with the following equation.

$$P_t = T_r \cdot n \times 1.047 \times 10^{-3}$$

$P_t$  : Transmission power (W)  
 $T_r$  : Torque (N·cm)  
 $n$  : Revolution (rpm)

**Table 2 Load correction factor ( $K_o$ )**

Load characteristics	Factor $K_o$
When the maximum load is used	1.0
When a normal load is used	1.3
When the frequency of starting and stopping is high	1.5

**Table 3 Pinion contact angle correction factors ( $K_{\theta_1}$ )**

Equation for contact angle calculation	$\Theta_1 = 180 - 57.3(D_p - d_p) / C$					
$(D_p - d_p) / C$	0.0	0.4	0.6	0.8	1.0	1.4
$\theta_1$ (°)	180	157	145	133	120	91
Correction factor $K_{\theta_1}$	1.00	0.94	0.91	0.87	0.82	0.70

**Table 4 Correction factors by initial stretch rate ( $K_t$ )**

$\Delta T$ (%)	4	5	6	7	8
Correction factor $K_t$	0.8	0.9	1.0	1.1	1.2

The correction factors in **Table 3** and **Table 4** correct the basic power rating; however, for convenience, they are in the form of correcting the transmission power.

### Step 3. Selecting a cross-sectional diameter

Select a cross-sectional diameter so that  $P_r \geq P_d$  from the basic power rating  $P_r$  and the design power  $P_d$  in Tables 5 to 8 (→ P. 300).

When  $P_r < P_d$ , increase the number of belts or increase the cross-sectional diameter.

### Step 4. Selecting an effective length

With the following equation, obtain the effective center perimeter of the belt and set the belt with the effective center perimeter closest to it from **Table 1** (→ P. 298).

$$L' = 2C' + 1.57(D_p + d_p) + \frac{(D_p - d_p)^2}{4C}$$

$L'$  : Effective center perimeter of the belt (mm)  
 $C'$  : Center distance (mm)  
 $D_p$  : Large pulley pitch diameter (mm)  
 $d_p$  : Pinion pitch diameter (mm)

Check the following points for the selected belt in accordance with the conditions.

#### (A) When the center distance is fixed

Check whether the initial stretch rate is between 4% and 8% with the following equation.

$$4 \leq \Delta t \leq 8$$

$\Delta t$  : Initial stretch rate  $100(L' / L - 1)$  (%)  
 $L'$  : Effective center perimeter of the belt (mm)  
 $L$  : Effective center perimeter of the standard belt (mm)

#### (B) When the center distance can be adjusted

Obtain the center distance with the following equation so that the initial stretch rate is 6%.

$$C = \frac{B + \sqrt{B^2 - 2(D_p - d_p)^2}}{4}$$

$$B = L \times 1.06 - 1.57(D_p + d_p)$$

$C$  : Center distance (mm)  
 $L$  : Effective center perimeter of the standard belt (mm)  
 $D_p$  : Large pulley pitch diameter (mm)  
 $d_p$  : Pinion pitch diameter (mm)

**Table 5 Table of basic power ratings for  $\phi 2$**

Pinion revolution (rpm)	Pinion pitch diameter (mm)				
	16	18	20	24	28
250	0.2	0.2	0.2	0.3	0.3
500	0.3	0.4	0.4	0.5	0.6
750	0.5	0.6	0.7	0.8	0.9
1000	0.7	0.8	0.9	1.1	1.2
1250	0.9	1.0	1.1	1.3	1.5
1500	1.0	1.2	1.3	1.5	1.8
1750	1.2	1.3	1.5	1.8	2.1
2000	1.4	1.5	1.7	2.0	2.3
2500	1.7	1.9	2.1	2.5	2.9
3000	2.0	2.3	2.5	2.9	3.3
3500	2.3	2.6	2.9	3.3	3.8

**Table 7 Table of basic power ratings for  $\phi 4$**

Pinion revolution (rpm)	Pinion pitch diameter (mm)				
	28	32	36	40	45
250	1.2	1.4	1.5	1.7	1.9
500	2.4	2.7	3.1	3.4	3.8
750	3.6	4.1	4.6	5.1	5.7
1000	4.8	5.4	6.1	6.8	7.6
1250	5.9	6.8	7.6	8.4	9.4
1500	7.1	8.1	9.0	9.9	11.3
1750	8.2	9.3	10.4	11.4	12.7
2000	9.3	10.5	11.7	12.8	14.2
2500	11.4	12.8	14.2	15.4	16.8
3000	13.4	14.9	16.3	17.5	18.8
3500	15.2	16.7	18.1	19.3	24.7

When the effective lengths in **Table 1** (P. 298) cannot be used, please contact us.

**Table 6 Table of basic power ratings for  $\phi 3$**

Pinion revolution (rpm)	Pinion pitch diameter (mm)				
	22	24	28	32	36
250	0.5	0.6	0.7	0.8	0.9
500	1.1	1.2	1.4	1.5	1.7
750	1.6	1.7	2.0	2.3	2.6
1000	2.1	2.3	2.7	3.1	3.4
1250	2.6	2.9	3.3	3.8	4.3
1500	3.2	3.4	4.0	4.5	5.1
1750	3.7	4.0	4.6	5.3	5.9
2000	4.2	4.5	5.3	5.9	6.6
2500	5.2	5.6	6.4	7.2	8.0
3000	6.1	6.6	7.5	8.4	9.2
3500	7.0	7.5	8.5	9.4	10.2

**Table 8 Table of basic power ratings for  $\phi 5$**

Pinion revolution (rpm)	Pinion pitch diameter (mm)				
	36	40	45	50	60
250	2.4	2.7	3.0	3.4	4.0
500	4.8	5.4	6.0	6.7	8.0
750	7.2	8.0	9.0	9.9	11.8
1000	9.5	10.6	11.8	13.1	15.5
1250	11.8	13.1	14.6	16.1	19.0
1500	14.1	15.5	17.3	19.0	22.1
1750	16.2	17.8	19.8	21.6	24.9
2000	18.7	20.0	22.1	24.0	27.4
2500	22.1	24.0	26.2	28.1	30.8
3000	25.5	27.4	29.3	30.8	31.9
3500	28.2	29.9	31.3	31.7	30.7

## Pulleys for the Bancollan Round Belt

As pulleys for the Bancollan round belt, please use pulleys with the following dimensions.

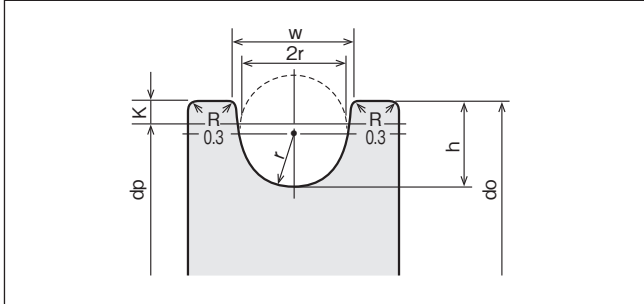


Table of pulley groove dimensions

(Unit: mm)

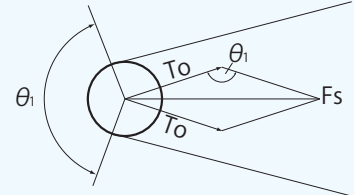
Belt cross-sectional diameter	φ2	φ3	φ4	φ5
Groove top width $W \pm 0.1$	2.2	3.2	4.2	5.2
Groove depth $h \begin{smallmatrix} +0.1 \\ -0 \end{smallmatrix}$	1.3	2.0	2.7	3.3
Groove bottom $r \pm 0.05$	0.9	1.4	1.9	2.4
Difference between outside diameter and pitch diameter $2k$	0.6	1.0	1.4	1.6
Minimum pulley pitch diameter	14	21	28	35

Avoid using V-grooved pulleys as they can cause partial abrasion of the belt.

## Shaft Load of Bancollan Round Belt

Obtain the shaft load of the Bancollan round belt with the following equation from the following table of the initial tension  $T_0$  in accordance with the initial stretch rate.

$$F_r = 2T_0 \sin(\theta_1/2)$$



Initial tension  $T_0$

(Unit: N)

Stretch rate (%) $\Delta t$	Belt cross-sectional diameter			
	φ2	φ3	φ4	φ5
4	1.08	2.45	4.31	6.76
5	1.27	3.53	6.57	10.3
6	1.57	4.31	7.64	11.8
7	1.76	4.80	8.62	14.4
8	1.96	5.19	9.60	16.5